# FR Gearing



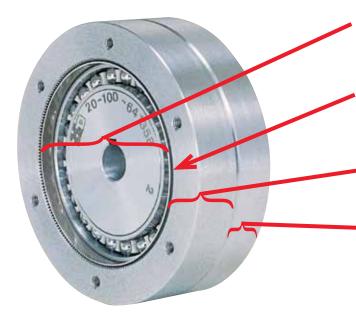


# Harmonic Drive gear

## **Contents**

The Basic Assembly	2
Configurations	3
Ordering Information	3
Typical Applications	4
Dimensions	5
Rating Table	6
Moment of Inertia	9
Torsional Characteristics and Backlash	10
Lubrication	11
Installed Relationship and Recommended Tolerances for Housing	12
Efficiency	13
No-Load Starting Torque and Backdriving Torque	15

# The Basic Assembly



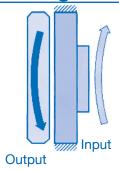
- 1) The Wave generator (WG) is a thin raced bearings assembly fitted onto an elliptical plug, and normally is the rotating input member.
- 2) The Flexspline (FS) is a non-rigid ring with external teeth on a slightly smaller pitch diameter than the Circular Spline. It is fitted over and is elastically deflected by the Wave Generator.
- 3) The Circular Spline (CS) is a rigid ring with internal teeth, engaging the teeth of the Flexspline across the major axis of the Wave Generator.
- (4) The Dynamic Spline (DS) is a rigid ring having internal teeth of same number as the Flexspline. It rotates together with the Flexspline and serves as the output member. It is identified by chamfered corners at its outside diameter.

The FR range has been developed to meet the trend towards flatter gear sets while retaining the many advantages of strain wave gearing. It consists of four main parts: Wave Generator, Flexspline, Dynamic Spline, and Circular Spline. When assembled, rotation of the Wave Generator as a high speed input member imparts a rotating elliptical shape to the Flexspline. This causes progressive engagement of its external teeth with the internal teeth of the Circular Spline. The

fixed Circular Spline, having a larger number of teeth than the Flexspline causes the latter to precess at a rate determined by the ratio of tooth difference to the total number of teeth.

With the same number of teeth as the Flexspline, The Dynamic Spline rotates with, and at the same speed as, the Flexspline and is the output member of the drive.

# **Configurations**



#### 1) Reduction Gearing

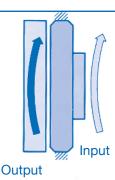
WG Input

CS Fixed

DS Output

Ratio as listed

Input and output counter rotate.



#### 2) Reduction Gearing

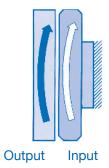
WG Input

**CS** Output

DS Fixed

Ratio  $\frac{1}{R+1}$ 

Input and output in same direction



#### 3) Reduction Gearing

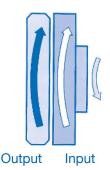
WG Fixed

**CS** Output

DS Input

Ratio  $\frac{R}{R+}$ 

Input and output in same direction.



#### 4) Differential

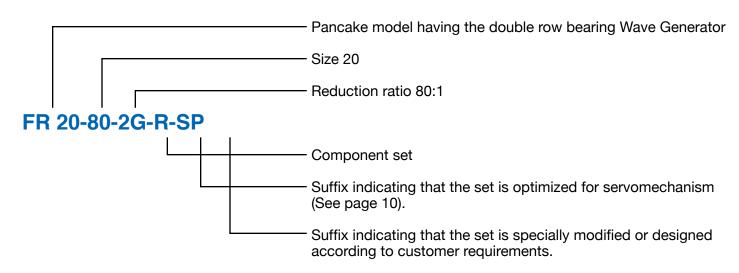
WG Control Input

CS Main-drive Input

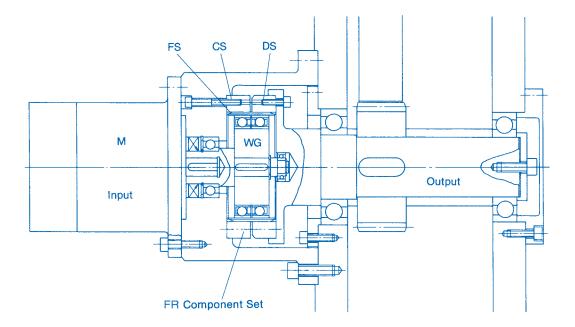
DS Main-drive Output

Numerous differential functions can be obtained by combinations of speeds and rotations on the three shafts.

# **Ordering Information**



# **Typical Applications**



Shown above is an FR component set used as a milling machine

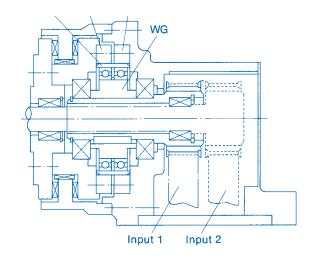
#### Advantages:

- Maximum reduction ratio in minimum space
- Low backlash
- Simple installation and maintenance

This application makes the most of the gear's unique features. The motor drives the Wave Generator (WG), and the Circular Spline (CS) is fixed to the casing. The output is taken from the Flexspline (FS) via the Dynamic Spline (DS). If the motor shaft is not oil sealed, an additional oil seal must be fitted.

The installation tolerances may be found on page 12.

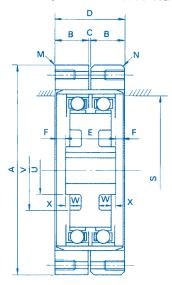
The FR component set shown is the drive of a robot joint. The input shaft is hollow, allowing another input shaft for the next stage to go through the Wave Generator. In another robotics application the hollow shaft is used to accommodate electric cables.

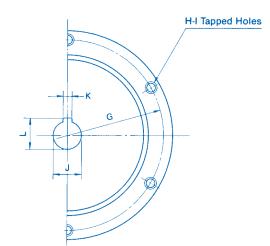


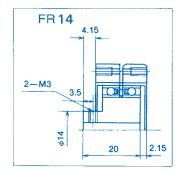


# **Dimensions**

#### FR 20-100 (See inset FR 14 Wave Generator)







F	-R	14	20	25	32	40	50	65	80	100
A	(h6)	50 <sup>0</sup> -0.016	70 -0.019	85 -0.022	110-0.022	135 -0.025	170-0.025	215-0.029	265-0.032	330-0.036
	В	8.5	12	14	18	21	26	35	41	50
	С	1	1	1	1	1	1	1	1	1
	D	18	25	29	37	43	53	71	83	101
	E	_	17.3	20	25.9	31.5	39.1	50.5	62	77.2
	F	_	3.85	4.5	5.55	5.75	6.95	10.25	10.5	11.9
	G	44	60	75	100	120	150	195	240	290
	н	6	6	6	6	6	6	6	8	8
	1	M3 x 6	M3 x 6	M4 x 8	M5 x 10	M6 x 12	M8 x 16	M10 x 20	M10 x 20	M12 x 24
	(H7)	6 <sup>+0.012</sup>	9 +0.015	<b>11</b> <sup>+0.018</sup>	<b>1</b> 4 <sup>+0.018</sup>	<b>1</b> 4 <sup>+0.018</sup>	<b>19</b> <sup>+0.021</sup>	24+0.021	28+0.021	28+0.021
J	max.	8	11	11	17	20	26	26	32	32
K (	JS9)	_	3±0.0125	<b>4</b> ±0.0150	5±0.0150	5±0.0150	6±0.0150	8±0.0180	<b>8</b> ±0.0180	8±0.0180
	L	_	10.4	12.8	16.3	16.3	21.8	27.3	31.3	31.3
M	(C)	0,2	0.2	0.2	0.2	0.4	0.4	0.4	0.4	0.4
N	(C)	1	1	1.5	1.5	1.5	1.5	1.5	2	2
	U	_	_	22	28	32	38	44	52	58
	V	_	_	32	42	52	62	86	100	128
,	W	_	_	4.8	6.1	7.6	9.8	12.6	16	19.7
	X	_	42	1.6	1.9	2.5	3.2	4.4	5.1	6.3
	s	29	42	53	69	84	105	138	169	211
Wt	lb	0.4	1.1	1.8	3.7	6.8	13.2	27.3	48.9	93.5
- wt	kgf	0.2	0.5	0.8	1.7	3.1	6.0	12.4	22.2	42.5

Solid & STINE axial location surfaces at D width must extend radially inward to at least to S for Flexspline containment. The surface hardness in the region where the Flexspline abuts is



Rating Table

Rat	ing	lab	<i>le</i>											
FR	Gear Ratio	Rated Speed	Rated T	orque at Orpm		ed Peak que		Average Torque		omentary que	Max. Inp	out Speed, rpm		average Input eed, rpm
	Hallo	rpm	N.m	ln.lb	N.m	In.lb	N.m	In.lb	N.m	In.lb	Oil Lub.	Grease Lub.**	Oil Lub.	Grease Lub.**
	50		4.4	39	5.4	48	5.4	48	13.7	121				
14	88	2,000	5.9	52	9.8	87	9.8	87	19.6	173*	6,000	3,600	4,000	2,500
'-	100	2,000	7.8	69	13.7	121	9.8	87	19.6	173*	0,000	0,000	4,000	2,000
	110		7.8	69	13.7	121	9.8	87	19.6	173*				
	50		25	221	34	301	34	301	69	611				
	80		34	301	41	363	41	363	72	637				
20	100	2,000	40	354	53	469	49	434	94	832	6,000	3,600	3,600	2,500
	128		40	354	67	593	49	434	102	903*				
	160		40	354	77	681	49	434	86	761*				
	50		39	345	55	487	55	487	108	956				
	80		56	496	69	611	69	611	122	1,080				
25	100	2,000	67	593	91	805	91	805	160	1,416	5,000	3,600	3,000	2,500
	120		67	593	108	956	108	956	190	1,682				
	160		67	593	135	1,195	108	956	172	1,522*				
	200		67	593	147	1,301	108	956	172	1,522*				
	50		76	673	108	956	108	956	216	1,912				
	78		108	956	137	1,212	137	1,212	245	2,168				
	100		137	1,212	176	1,558	176	1,558	323	2,859				
32	131	2,000	137	1,212	255	2,257	216	1,912	451	3,991	4,500	3,600	2,500	2,300
	157		137	1,212	294	2,602	216	1,912	500	4,425*				
	200		137	1,212	314	2,779	216	1,912	372	3,292*				
	260		137	1,212	314	2,779	216	1,912	372	3,292*				
	50		137	1,212	196	1,735	196	1,735	353	3,124				
	100		196 255	1,735 2,257	245 314	2,168 2,779	245 314	2,168 2,779	431 549	3,814 4,859			2,000	
40	128	2,000	294	2,602	392	3,469	392	3,469	686	6,071	4,000	3,300		2,000
40	160	2,000	294	2,602	461	4,080	451	3,991	813	7,195	4,000	3,300		2,000
	200		294	2,602	529	4,682	451	3,991	745	6,593*				
	258		294	2,602	627	5,549	451	3,991	745	6,593*				
	80		363	3,213	440	3,894	441	3,903	784	6,938				
	100		470	4,160	578	5,115	578	5,115	1,019	9,018				
	120		559	4,947	696	6,160	696	6,160	1,225	10,841				
50	160	1,700	559	4,947	833	7,372	833	7,372	1,470	13,010	3,500	3,000	1,700	1,700
	200		559	4,947	960	8,496	843	7,461	1,411	12,487*				
	242		559	4,947	1176	10,408	843	7,461	1,411	12,487*				
	78		745	6,593	921	8,151	921	8,151	1,617	14,311				
	104		1,070	9,470	1,340	11,859	1,340	11,859	2,360	20,886	-			
	132		1,070	9,470	1,650	14,603	1,570	13,895	2,890	25,577				
65	158	1,400	1,070	9,470	1,970	17,435	1,570	13,895	3,450	30,533*	3,000	2,200	1,400	1,400
	208	1	1,070	9,470	2,180	19,293	1,570	13,895	2,590	22,922*				
	260		1,070	9,470	2,200	19,470	1,570	13,895	2,590	22,922*				
	80		1,320	11,682	1,640	14,514	1,640	14,514	2,870	25,400				
	96		1,660	14,691	2,050	18,143	2,050	18,143	3,580	31,772				
	128		2,300	20,355	2,820	24,957	2,830	25,046	4,960	43,896				
80	160	1,200	2,360	20,798	3,380	29,913	3,130	27,701	5,940	52,569	2,500	2,000	1,200	1,200
	194		2,350	20,798	4,300	38,055	3,130	27,701	6,900	61,065*				
	258		2,350	20,798	4,350	38,498	3,130	27,701	5,170	45,755				
	320		2,350	20,798	4,350	38,498	3,130	27,701	5,170	45,755*				
	80		2,350	20,621	2,870	25,400	2,870	25,400	5,040	44,604				
	100		2,330	28,320	3,940	34,869	3,940	34,869	6,920	61,242				
	120		3,200	34,427	4,780	42,303	4,780	42,303	8,400	74,340				
100	160	1,000	4,470	39,560	6,230	55,136	5,720	50,622	10,950	96,908	2,000	1,700	1,000	00 1,000
	200		4,470	39,560	7,090	62,747	5,720	50,622	12,440	110,094				
	242		4,470	39,560	7,960	70,446	5,720	50,622	9,410	83,279*				
	320		4,470	39,560	7,960	70,446	5,720	50,622	9,410	83,279*				

 $<sup>^{\</sup>star}$ Torque value limited by "Ratcheting", see page 8.  $^{\star\star}$ For operating conditions see grease lubrication, page 11.

# How To Use The Rating Table

Because of their simple, convenient construction and positional and rotational accuracy, Harmonic Drive™ component sets are used in large numbers in servo-controlled drives where the load and driving speed are seldom constant. With such applications in mind, the rating table presents four important torque capacity limits:

- Rated Torque at rated speed
- Allowable limit for Average Torque
- Allowable limit for repeated Peak Torque
- Allowable limit for Momentary Peak Torque

#### Rated Toque at Rated Input Speed

This is the maximum allowable output torque that can be developed continuously at the rated input speed shown in the first column. When a Harmonic Drive<sup>TM</sup> FR product is to be operated at speeds other than the rated torque, the actual torque load at each speed needs to be converted to an Equivalent Torque ( $T_{eq}$ ) using formulas 1, 2, and 3 on page 9.

#### Allowable Limit for Average torque

When a Harmonic Drive™ FR product is used under a variable load, Average Torque may be calculated by formula 1 on page 9. Thus calculated Average Torque should not exceed this limit. Ignoring this limit will result in excessive heat generation, tooth wear, and deterioration of lubricant.

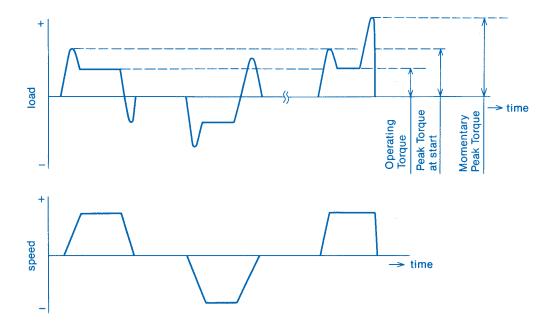
#### Allowable Limit for Repeated Peak Torque

This is the allowable output torque that can be developed when starting and stopping operation. Peak torque developed at starting and stopping can be estimated if the moment of inertia of motor and load, and accelerating (or decelerating) time are known.

#### Allowable Limit for Momentary Peak torque

Aside form the peak torque developed at starts and stops, FR unit transmission may be subjected to yet another type of peak torque, an example of which is the shock load generated by an emergency stop of the servo system. Another example is the shock load generated when the driven element, such as a robot hand, accidentally hits a hard object. The Rating Table gives the limit for such Momentary Peak Torque. *Momentary Peak Torque must be restricted not only in its value but also in its frequency of occurrence.* This is further explained in the next chapter titled "Factors Affecting Torque Capabilities."

The variation of load and speed in typical operating cycles are schematically illustrated in the diagrams below.





# **Factors Affecting Torque Capabilities**

The torque capabilities quoted in the Rating Table are determined by considering three factors: the strength of the Flexspline, the unit's ability to resist a phenomenon called ratcheting, and the operating life of the Wave Generator bearing.

#### Flexspline Strength

The Flexspline, the non-rigid member, is subjected to repeated deflections, and its strength limits the rating of the entire unit. The values given for Rated Torque at Rated Speed, Limits for Average Torque, and repeated Peak Torque are for the infinite fatigue life of the Flexspline. The Limit For Momentary Peak torque, however, exceeds the infinite fatigue limit of the Flexspline, and the load at this level must be restricted to 1x10<sup>4</sup> flex-cycles. *Operating conditions exceeding the limit will lead to failure of the Flexspline.* 

The allowable frequency of occurrence (N) of Momentary Peak Torque may be calculated by:

$$N \le \frac{1 \times 10^4}{2 \times \frac{n}{60} \times t}$$

where n is the input speed in rpm and t is the duration of Momentary Peak Torque in seconds. Note that there are two flex-cycles for each input revolution. For example, if Momentary peak torque is expected to occur at a 2,00 rpm input speed and to last for 0.15 seconds, then:

$$N \le \frac{1 \times 10^4}{2 \times \frac{2,000}{60} \times 0.15} \le 1,000$$

#### Ratcheting

Another torque limit for under dynamic load is set by a momentary load releasing phenomenon called ratcheting. This is slippage of the meshing of the Circular Spline and Flexspline teeth as the result of deformation of drive members under excessive load. In the Rating Table, values marked with asterisk(\*) are the torque limits related to ratcheting. Ratcheting usually does not lead to immediate drive failure, but may result in "dedoidal", an improper tooth mesh that may cause undesirable vibration or eventual failure of the Flexspline. The ratchet point will decrease with each subsequent incident, and should not be intentionally utilized for machine overload protection purposes.

#### Operator Life of the Wave Generator Bearing

The life expectancies of Harmonic Drive™ component sets are based on the median life of the Wave Generator bearing. Rated Torque at Rated Speeds are based on a median bearing life (L₅₀) of approximately 15,000 hours. This means that operated at the constant rated speed and rated torque, 50 percent of a group of bearings would fail before these service hours.

Life expectancy of a unit operated at speed n and torque T conditions may be estimated by the relationship given below:

$$L_{50}=15,000 \bullet \frac{\text{Rated Speed}}{\text{n}} \bullet \left(\frac{\text{Rated torque a rated Speed}}{\text{T}}\right)^{3}$$

#### **Operating Speed**

The Rating Table gives two limits, Maximum Input Speed and Limit for Average Input Speed, for both oil-lubricated and grease-lubricated units. Units may be operated continuously up to the maximum input speed limits, provided that the average speed during the operation, as calculated by formula 2, on pages 11 and 12, does not exceed the limit for the average input speed.

For continuous operation of grease-lubricated units at these speed limits, it is recommended that grease specially developed for Harmonic Drive™ gears be used. For a list of recommended available lubricants, see page 10 and 11.

# How To Select A Unit From The Rating Table

In actual applications, operating conditions are seldom constant. Torque and speed vary, and peak torque develops during acceleration and deceleration. The first step is to convert these variable torques and speeds to a constant average torque and speed.

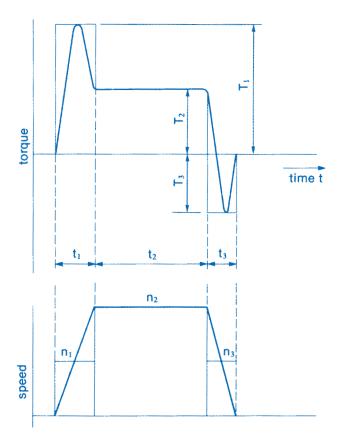
#### Step 1: Average Torque and Speed

Consider a hypothetical operating cycle such as shown in the diagram on the following page

 $T_1$  is the Peak torque developed during the acceleration  $t_1$ ,  $T_2$  is the Operational torque at constant speed  $n_2$  that lasts for a period of  $t_2$ , and  $T_3$  is the Peak Torque developed during acceleration  $t_3$ .  $n_1$  and  $n_3$  represent the Average Speed during acceleration and deceleration, which in this case are  $n_2$ 



2



The Average Torque T<sub>av</sub> for this cycle is calculated:

Tav= 
$$\sqrt[3]{\frac{n_1t_1T_1^3+n_2t_2T_2^3+n_3t_3T_3^3}{n_1t_{1+}n_2t_{2+}n_3t_3}}$$

The Average Speed nav for this cycle is

$$n_{av} = \frac{n_1 t_1 + n_2 t_2 + n_3 t_3}{t_1 + t_2 +_3}$$

#### Step 2

The second step is to calculate as below an Equivalent Torque Teq that would give a median bearing life of 15,000 hours.

$$T_{eq} = T_{av} \bullet \sqrt[3]{\frac{n_{av}}{Rated Speed}}$$

#### Step 3

The third step is to estimate the level of Momentary Peak Torque and the frequency of its occurrence. Estimated Momentary Peak Torque must be less than the allowable limit given in the Rating Table, and the frequency of its occurrence must not exceed 1x10<sup>4</sup> flex-cycles.

#### Step 4

The fourth step is to check these operating conditions against the values and limits given in the Rating Table for a unit under consideration. If any of the operating conditions exceed the limits, either the operating condition must be relaxed or another unit must be selected.

# Toll Free Phone (877) SERV098 Toll Free Fax (877) SERV099 www.electromate.com sales@electromate.com

#### Example

A drive unit FR 40 ratio 100:1 is being considered for these operating torques and speeds:

#### **Torque**

<ul> <li>Peak torque during</li> </ul>	acceleration T <sub>1</sub> 2500	) lb-in
<ul> <li>Operating torque T<sub>2</sub></li> </ul>	<sub>2</sub> 1740	) lb-in

• Peak torque during deceleration T₃.....2200 lb-in

#### Time

Accelerating time t <sub>1</sub>	0.2 seconds
Operating time t <sub>2</sub>	2.0 seconds

#### Speed

- Operating speed n<sub>2</sub> ......2200 rpm
- Average speed during acceleration and deceleration n₁ and n₃ ...... 1100 rpm

#### Step 1

$$T_{av} = \sqrt[3]{\frac{1100 \times 0.2 \times 2500^3 + 2200 \times 2 \times 1740^3 + 1100 \times 0.3 \times 2200^3}{1100 \times 0.2 + 2200 \times 2 + 1100 \times 0.3}}$$

=1826 lb-in

$$n_{av} = \frac{1100 \times 0.2 + 2200 \times 2 + 1100 \times 0.3}{0.2 + 2 + 0.3} = 1980 \text{ rpm}$$

#### Step 2

Teq=1826• 
$$\sqrt[3]{\frac{1,980}{2,000}}$$
 1,820 lb-in

#### Step 3

It is estimated that approximately 4,500 lb-in of peak torque will develop momentarily at an emergency stop, which is estimated to take place at 2,200 rpm and last for 0.15 seconds. Allowable frequency of such Momentary Peak Torque is:

$$N = \frac{1 \times 10^4}{2 \times \frac{2,200}{60} \times 0.15} = 900 \text{ times}$$

#### Step 4: Conclusion

Entering the Rating Table at the line corresponding to FR 40-100 we find that T<sub>eq</sub>, T<sub>av</sub>, Repeated Peak Torque and Momentary Peak Torque values, and speed conditions of this application are within the allowable limits:

	Estimated Operating Conditions	Rating and Limits
Equivalent Torque T <sub>eq</sub>	1,820 lb-in	2,220 lb-in
Average Torque, Tav	1,826 lb-in	2,780 lb-in
Repeated Peak Torque	2,500 and 2,200 lb-in	2,770 lb-in
Momentary Peak Torque	4,500 lb-in	4,860 lb-in
Average Speed, nav	1,980 rpm	2,000 rpm

Therefore, this unit is considered adequate for this application. It can be lubricated either with oil or Harmonic Grease SK-1. The occurrence of emergency stops must be less than 900 times during the expected service life of this unit.

# **Moment of Inertia**

Inertia values at the high-speed shaft for FR component sets are given in table below.

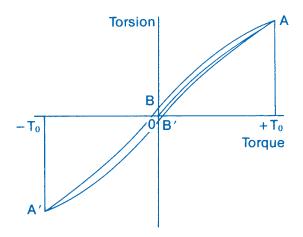
Units with reduced input inertia may be supplied on a custom basis. Since the reduction of inertia

is achieved by modifying the Wave generator and may affect the unit's torque capacity, each application requiring reduced inertia needs to be evaluated individually.

F		14	20	25	32	40	50	65	80	100
Moment of	kg-cm²	0.06	0.32	0.70	2.6	6.8	21	76	213	635
Inertia J	lb-in²	0.02	0.11	0.24	0.89	2.3	7.1	26	73	217

### Torsional Characteristics and Backlash

The torsional characteristics of the drive are an important consideration in a servo-drive system. When torque is applied to the output of the gear unit with the input rotationally locked, the torque-torsion relationship measured at the output typically follows the loop O-A-B-A'-B'-A illustrated.



Three major characteristics of interest in a servo driver are:

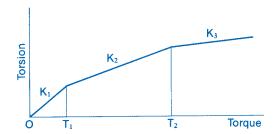
- 1) The gear exhibits soft wind-up characteristics in the low torque region. The spring rates for three regions of the torque-torsion curve are given on pages 9 and 10.
- 2) The backlash in an assembled component set is defined as pure play existing between the input and output. Because of double row teeth mesh, FR component set exhibits backlash somewhat larger than CSF component sets. Backlash of Series R (for robotics application) is optimized below 1.5 minutes of arc for all sizes. Backlash ofnon-optimized units are quoted below. Values are measured at output with input looked.

Backlash in assembled FR									of arc
FR	14	20	25	32	40	50	65	80	100
Optimized, less than	1.5	1.5	1.5	1.5	1.5	1.5	1.5	1.5	1.5

#### Spring Rate

For servo-drive applications, the torsional stiffness of the FR component set may be evaluated by dividing the torque-torsion curve in three major regions: a small torque region  $O-T_1$ , a middle torque region  $T_1-T_2$ , and a linear region  $T_2-T_3$ . Spring rates of these regions vs. size standard units are tabulated. The values quoted are the average of many tests of actual units. The spring rate of an individual unit may vary within approximately  $\pm 30\%$  of the average.

#### **Torque-Torsion Curve**



K<sub>1</sub>: Spring rate for torque region O-T<sub>1</sub>

 $K_2$ : Spring rate for torque region  $T_1$ - $T_2$ 

K₃: Spring rate for torque region over T₂



#### Spring Rate FR Component Sets Series R

FR	1	ri 💮	K₁x1	04/rad	1	Γ2	K₂x1	0⁴/rad	K₃x10	)⁴/rad
FR	Nm	lb-in	Nm	lb-in	Nm	lb-in	Nm	lb-in	Nm	lb-in
14	1.0	8.9	0.26	2.3	5.0	44	0.64	5.7	1.0	8.9
20	2.0	17.4	0.81	7.2	23	200	2.8	25	3.0	27
25	5.0	43	1.6	14	37	330	3.7	33	5.4	48
32	9.0	78	2.8	25	78	690	8.4	74	12	106
40	18	156	6.1	54	156	1380	18	159	24	212
50	34	304	12	106	294	2600	34	300	48	425
65	78	694	26	230	549	4860	64	566	84	743
80	147	1300	51	451	1078	9550	130	1150	165	1460

#### **Calculation Example**

This formula is used to find the total wind-up ( $\theta$ ) at the output of an FR 25 ratio 100:1, when a torque load of 600 lb-in is applied.

$$\theta = \frac{T_1}{K_1} + \frac{T_2 - T_2}{K_2} + \frac{600 - T_2}{K_3} + Backlash$$

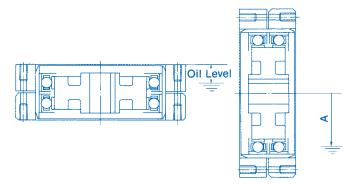
$$= \frac{43}{14 \times 10^4} + \frac{330 - 43}{33 \times 10^4} + \frac{600 - 330}{48 \times 10^4} + \text{Backlash}$$

$$= 17.4 \times 10^{-4} \text{ rad} + \text{Backlash}$$

## Lubrication

As with CSF component sets, oil lubrication is preferred. Although FR sets may be used in any attitude, it is essential that the wave generator bearing and gear teeth be properly lubricated. Minimum required oil amounts are tabulated below, but the actual amount will depend on the size of the housing.

In the case of horizontal shaft installation, the appropriate oil level below the gear set center line must be maintained as tabulated. For vertical mounting, the recommended level is the center line of



the Upper Wave generator bearing balls.

	R									
Α	mm	7	12	15	31	38	44	62	75	94

#### **Grease Lubrication**

FR component sets may be operated with grease lubrication at rated torque but at reduced duty cycle. Imperial Molub Alloy No. 2 is recommended.

Grease lubrication may be used in applications where the duty cycle is less than 10 percent time-on and the length of time-on does not exceed 10 minutes of continuous operation. The maximum input speeds allowable for units lubricated by Molub Alloy No. 2 are given below:

-	FR									
	S	24.5	38	48	63	76	95	126	152	190

#### Harmonic Grease HC-1

As a result of an extensive search for a grease that will render improved performance in Harmonic Drive<sup>™</sup> gears, a new grease, named Harmonic Grease HC-1, has been developed. With Harmonic Grease HC-1, *units may be operated continuously.* 

An important consideration in grease lubrication is ensuring that as much grease as possible is retained where lubrication is needed. To achieve this, it is recommended that the axial location surfaces at D width extend radially inward to the dimension shown. However, such an extension is not recommended for oil lubrication.

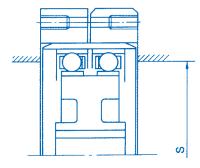
#### Oil Level and Minimum Oil Quantity

#### Oil Temperature

In normal use, the oil temperature must not exceed 90°C, as oil loses its lubricating capability quickly above this limit.

#### Oil Change

The first change should be performed after 100 hours of operation. The need to perform subsequent oil changes will depend on operating conditions, but should take place at intervals of approximately 1,000 running hours.



									mm
FR	14	20	25	32	40	50	65	80	100
S	24.5	38	48	63	76	95	126	152	190

#### **Grease Changes**

When operating the FR at rated torque, change grease after about 1,00 running hours. Light duty operation may permit longer service intervals. To change grease, Completely disassemble and clean units before re-greasing. Apply grease generously inside the Flexspline, the Wave generator bearing,

and teeth of both the Circular and Dynamic Splines and the Flexspline.

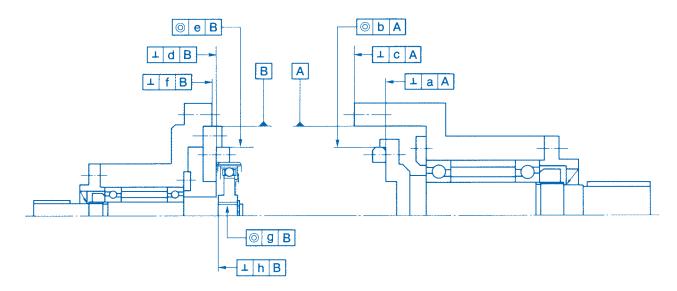
The approximate amount of grease needed for each Harmonic Drive<sup>™</sup> FR Component Set is tabulated below:

FR		14	20	25	32	40	50	65	80	100
Grease Weight	gr.	8	18	30	60	100	150	320	570	1150
	oz.	0,3	0.6	1	2	3.5	5.5	11	20	40

# Installed Relationship and Recommended Tolerances for Housing

The Dynamic Spline is distinguished by its chamfered outer edge. FR component sets may be operated in any attitude with suitable bearing support of the input and output shaft, and a means of fixing the Circular

Spline against rotation. Recommended installed relationships for the Series R units, (recommended for robotics application) are shown below:

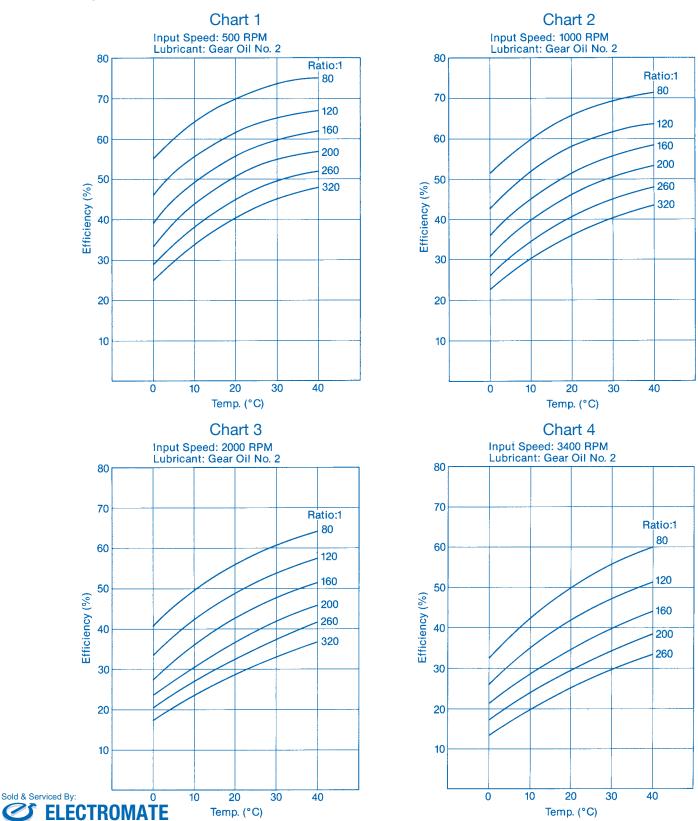


FR	а	b	С	d	е	f	g	h
14	0.013	0.015	0.016	0.013	0.015	0.016	0.011	0.007
20	0.017	0.016	0.020	0.017	0.016	0.020	0.013	0.010
25	0.024	0.016	0.029	0.024	0.016	0.029	0.016	0.012
32	0.026	0.017	0.031	0.026	0.017	0.031	0.016	0.012
40	0.026	0.019	0.031	0.026	0.019	0.031	0.017	0.012
50	0.028	0.024	0.034	0.028	0.024	0.034	0.021	0.015
65	0.034	0.027	0.041	0.034	0.027	0.041	0.025	0.015
80	0.043	0.033	0.052	0.043	0.033	0.052	0.030	0.015
100	0.057	0.038	0.068	0.057	0.038	0.068	0.035	0.015

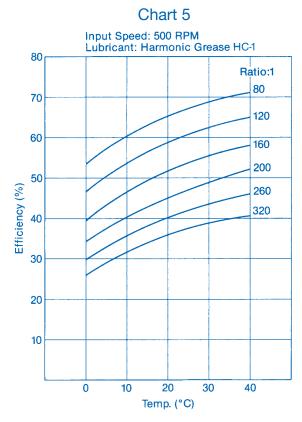
# **Efficiency**

Efficiency varies depending on input speed, ratio, load level, temperature, and type of lubrication. The effect of these factors are illustrated in the curves shown.

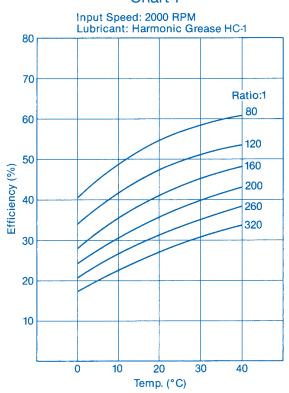
#### Efficiency vs. Speed, Temperature, Reduction Ratio, and oil Lubrication



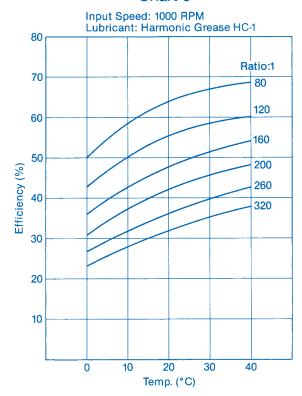
Toll Free Phone (877) SERV098 Toll Free Fax (877) SERV099 www.electromate.com sales@electromate.com



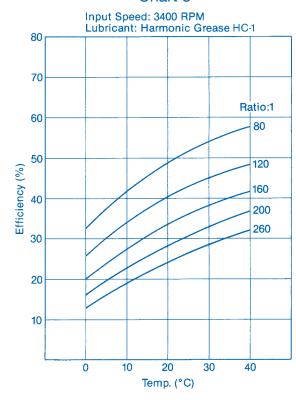




#### Chart 6



#### Chart 8



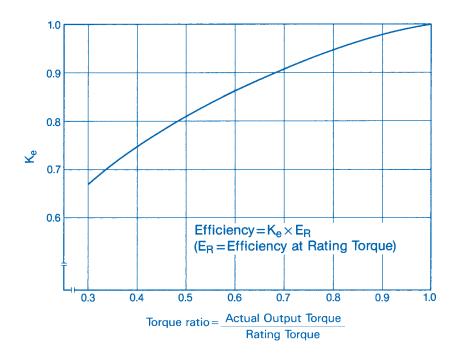


Toll Free Phone (877) SERV098 Toll Free Fax (877) SERV099 www.electromate.com sales@electromate.com

# Efficiency vs. Load

Efficiency of the gears vary depending on output torque. The efficiency curves given on the preceding pages are for units operating at an output torque

rated for 2,00 rpm. Efficiency of a unit operating at a load below the rated torque may be estimated using a compensation curve and formula shown below.



Ex. Efficiency of an FR 40-160-2GR operating at an input speed of 1,00 rpm, output torque of 1,560 lb-in, and at 40°C may be estimated as follows:

Torque ratio=
$$\frac{1,560}{2,600} = 0.6$$

 $K_{e} = 0.87$ 

Efficiency (at 1,560 lb-in) =  $58 \times 0.87 = 50\%$ 

# No-Load Starting Torque and Backdriving Torque

FR		14	20	25	32	40	50	65	80	100
Starting Torque	Ncm	0.7~4	0.7~6	0.7~20	1~30	3~50	4~100	7~200	28~280	98~680
	oz-in	1~5	1~8	1~28	1.4~42	4.2~70	5~140	10~280	40~400	140~970
Backdriving Torque	Nm	0.7~10	1~19	3~48	4~80	7~190	15~340	30~480	48~790	290~2500
	lb-in	6~90	9~170	26~430	35~700	60~1700	130~3000	260~4300	430~7000	2600~22000

Values quoted are based on actual tests with the component sets assembled in their housings, and inclusive of friction resistance of oils seals, and churning of oil.

sales@electromate.com